

AE/ME 339 Computational Fluid Dynamics (CFD)

K. M. Isaac Professor of Aerospace Engineering Eqs. 6.96 and 6.97 are the x and y momentum equations written in terms of the velocity and pressure correction terms.

The next important step is to get an expression for pressure correction by using the condition that the velocity field must satisfy the conservation of mass equation.

The p' formula that will be used is not an exact representation. It is devised such that when convergence is achieved:

 $p' \rightarrow 0$

and the formula for p' tends to the physically correct continuity equation.

Patankar sets $A', B', (\rho u')^n, (\rho v')^n$ to zero in Eqs. 6.96 and 6.97 which would yield the following equations.

Recall that

$$(\rho u')_{i+1/2,j}^{n} = (\rho u)_{i+1/2,j}^{n} - (\rho u^{*})_{i+1/2,j}^{n}$$

Eq. 6.98 can be written as

Similarly, Eq. 6.99 becomes

The continuity equation centered around the point (i,j) using central differencing (CD) becomes

$$\frac{(\rho u)_{i+1/2,j} - (\rho u)_{i-1/2,j}}{\Delta x} + \frac{(\rho v)_{i,j+1/2} - (\rho v)_{i,j-1/2}}{\Delta y} = 0.....(6.102)$$

12/21/01

Topic9_Pressure_Correction

Substitute Eqs. (6.100) and and (6.101) in Eq. (6.102) and dropping the superscript gives

$$\frac{\left(\rho u^{*}\right)_{i+1/2,j} - \Delta t / \Delta x \left(p_{i+1,j}^{'} - p_{i,j}^{'}\right) - \left(\rho u^{*}\right)_{i-1/2,j} + \Delta t / \Delta x \left(p_{i,j}^{'} - p_{i-1,j}^{'}\right)}{\Delta x}$$

$$\frac{\left(\rho v^{*}\right)_{i,j+1/2} - \Delta t / \Delta y \left(p_{i,j+1}^{'} - p_{i,j}^{'}\right) - \left(\rho v^{*}\right)_{i,j-1/2} + \Delta t / \Delta y \left(p_{i,j}^{'} - p_{i,j-1}^{'}\right)}{\Delta y} = 0....(6.103)$$

Eq. (6.103) can be rearranged to give (see next slide for expressions for a, b, c and d)

$$ap'_{i,j} + bp'_{i+1,j} + bp'_{i-1,j} + cp'_{i,j+1} + cp'_{i,j-1} + d = 0.....(6.104)$$

12/21/01

Computational Fluid Dynamics (AE/ME 339)K. M. IsaacPressure Correction MethodMAEEM Dept., UMR

Where a, b, c and d are given by the following expressions

$$a = 2\left\lfloor \frac{\Delta t}{\left(\Delta x\right)^2} + \frac{\Delta t}{\left(\Delta y\right)^2} \right\rfloor \qquad b = -\frac{\Delta t}{\left(\Delta x\right)^2} \qquad c = -\frac{\Delta t}{\left(\Delta y\right)^2}$$

$$d = \frac{1}{\Delta x} \left[\left(\rho u^* \right)_{i+1/2,j} - \left(\rho u^* \right)_{i-1/2,j} \right] + \frac{1}{\Delta y} \left[\left(\rho v^* \right)_{i,j+1/2} - \left(\rho v^* \right)_{i,j-1/2} \right]$$

Eq. (6.104) gives the pressure correction.

The SIMPLE algorithm:

The pressure correction formula (Eq. 6.104) is approximate because we set $A', B', (\rho u')^n, (\rho v')^n$ equal to zero. Hence the term "Semi-implicit" in the name.

This makes the pressure effect to be localized.

for the staggered grid given in Figure 6.15

- 1. Guess $(p^*)^n$ at all pressure nodes and set $(\rho u^*)^n, (\rho v^*)^n$ arbitrarily at the appropriate velocity nodes.
- 2. Solve for $(\rho u^*)^{n+1}$, $(\rho v^*)^{n+1}$ using Eqs. (6,94) and (6.95) respectively.
- 3. Substitute these values of $(\rho u^*)^{n+1}$, $(\rho v^*)^{n+1}$ in Eq. (104) and solve for p' at the interior nodes (boundary nodes will be treated separately). Relaxation procedure would work.
- 4. Calculate $p^{n+1} = (p^*)^n + p'$ at all nodes.
- 5. The values of (p)n+1 obtained in the previous step are used for solving the momentum equations.
- 6. Repeat steps 2-5 until convergence criteria are satisfied.

The superscript (n) and (n+1) used in the above equations are pseudo-time in the sense that solution obtained from this procedure will not be "**time-accurate**."

Therefore, the method essentially is a "**time-dependent**" method for steady state problems.

(n) and (n+1) therefore, can be thought of as representing sequential iteration steps.

The above procedure can cause the solution to diverge. Extensive use of "under-relaxation factors" is employed as a remedy. The following equation is an example of how under-relaxation factor can be used.

$$p^{n+1} = p^n + \alpha_p p'.....(6.106)$$

where a is the under relaxation factor.

$$0 \le \alpha_p \le 1$$

12/21/01

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Boundary conditions for pressure correction method (6.8.6)



At the inflow boundary:

p and v are specified, u is allowed to float. Therefore, p' = 0 at the inflow boundary.

<u>Outflow boundary:</u> p is specified and u and v are allowed to float.

<u>At the walls:</u> No slip condition gives all velocities to be zero. The y-momentum (Eq. 6.79) equation at the wall can be written as:

Since v = 0 at the wall, the first term on the RHS will be zero. Also we approximate the second term to be zero because it is usually small.

12/21/01

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Therefore, we have the following approximate condition at the wall

$$\left(\frac{\partial p}{\partial y}\right)_{w} = 0.....(6.108)$$



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