



AE/ME 339
Computational Fluid
Dynamics (CFD)

K. M. Isaac

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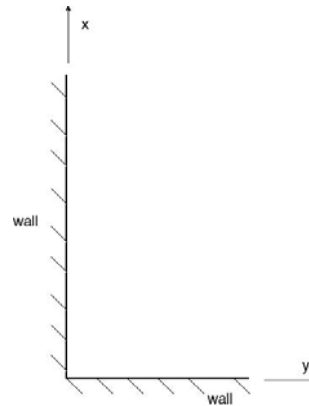
topic11_heated_flow

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Computational Fluid Dynamics (AE/ME 339)

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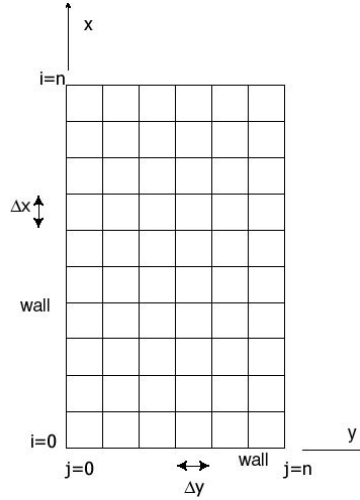
Let us consider flow over a heated vertical wall as shown in figure. A free convection flow induced by buoyancy forces is established close to the wall.



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Boundary conditions:

At $t = 0$:

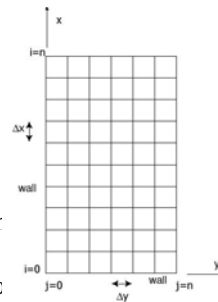
$u = v = 0, T = T_{inf}$ (everywhere in the solution domain)

At $t > 0$:

$y = 0: u = v = 0, T = T_w$ (heated wall condition)

$y = inf: u = 0, T = T_{inf}$ (far field condition)

$x = 0: u = v = 0, T = T_{inf}$ (bottom wall conditic)



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Governing equations

$$\frac{\partial u}{\partial x} + \frac{\partial v}{\partial y} = 0 \dots \dots \dots (1)$$

$$\frac{\partial u}{\partial t} + u \frac{\partial u}{\partial x} + v \frac{\partial u}{\partial y} = -\frac{1}{\rho} \frac{\partial p}{\partial x} + g_x \beta (T - T_\infty) + \nu \left(\frac{\partial^2 u}{\partial x^2} + \frac{\partial^2 u}{\partial y^2} \right) \dots \dots \dots (2)$$

$$\frac{\partial v}{\partial t} + u \frac{\partial v}{\partial x} + v \frac{\partial v}{\partial y} = -\frac{1}{\rho} \frac{\partial p}{\partial y} + g_y \beta (T - T_\infty) + \nu \left(\frac{\partial^2 v}{\partial x^2} + \frac{\partial^2 v}{\partial y^2} \right) \dots \dots \dots (3)$$

$$\frac{\partial T}{\partial t} + u \frac{\partial T}{\partial x} + v \frac{\partial T}{\partial y} = \frac{k}{\rho c_p} \left(\frac{\partial^2 T}{\partial x^2} + \frac{\partial^2 T}{\partial y^2} \right) + \frac{\mu}{\rho c_p} \left[2 \left(\frac{\partial u}{\partial x} \right)^2 + 2 \left(\frac{\partial v}{\partial y} \right)^2 + \left(\frac{\partial u}{\partial x} + \frac{\partial v}{\partial y} \right)^2 \right] \dots \dots \dots (4)$$

The Navier-Stokes equations derived in Chapter 2 can be simplified using the boundary layer assumptions.

$$\frac{\partial u}{\partial x} + \frac{\partial v}{\partial y} = 0 \dots \dots \dots (1)$$

$$\frac{\partial u}{\partial t} + u \frac{\partial u}{\partial x} + v \frac{\partial u}{\partial y} = g_x \beta (T - T_\infty) + \nu \frac{\partial^2 u}{\partial y^2} \dots \dots \dots (5)$$

$$\frac{\partial T}{\partial t} + u \frac{\partial T}{\partial x} + v \frac{\partial T}{\partial y} = \frac{k}{\rho c_p} \frac{\partial^2 T}{\partial y^2}$$

$$\frac{\partial T}{\partial t} + u \frac{\partial T}{\partial x} + v \frac{\partial T}{\partial y} = \alpha \frac{\partial^2 T}{\partial y^2} \dots \dots \dots (6)$$

Note the neglected terms in the above equations

Eq. 2: 2nd derivative of u w.r.t x is neglected because it is small compared to the other term.
Pressure gradient is assumed to be zero.

Eq. 3: This equation is not solved in the present formulation
All the terms on the RHS are assumed to be zero.
Since p is assumed to be a constant, there is no need to use this equation.

Eq. 4: 2nd derivative of temperature w.r.t x and the viscous dissipation terms are neglected.

The simplified equations (continuity, x-momentum and energy) can be non-dimensionalized as follows:

$$\xi = x(g\beta\Delta T / \nu^2)^{\frac{1}{3}}$$

$$\eta = y(g\beta\Delta T / \nu^2)^{\frac{1}{3}}$$

$$\tau = t(g\beta\Delta T)^{\frac{2}{3}} / \nu^{\frac{1}{3}}$$

$$\bar{u} = u / (g\beta\Delta T)^{\frac{1}{3}}$$

$$\bar{v} = v / (g\beta\Delta T)^{\frac{1}{3}}$$

$$\theta = \frac{T - T_{\infty}}{T_1 - T_{\infty}} = \frac{T - T_{\infty}}{\Delta T}$$

Note that g_x is the magnitude of the gravity component in the x-direction. β is the volumetric coefficient of thermal expansion defined as

$$\beta = -\frac{1}{\rho} \frac{\partial \rho}{\partial T}$$

Further, we assume that the Prandtl number $Pr (= \mu c_p / k = \nu / \alpha) = 1$. The equations will now be obtained in a form that is easy to solve numerically.

Note β has units of (1/K)

If the assumption $Pr = 1$ is not made, the RHS of the energy equation will have a Pr factor.

The energy equation for non-unity Prandtl number can be differenced using the explicit formulation as follows

$$\frac{\theta_{i,j}^{n+1} - \theta_{i,j}^n}{\Delta \tau} + \bar{u}_{i,j}^n \frac{\theta_{i,j}^n - \theta_{i-1,j}^n}{\Delta \xi} + \bar{v}_{i,j}^n \frac{\theta_{i,j+1}^n - \theta_{i,j}^n}{\Delta \eta} = \frac{1}{Pr} \left(\frac{\theta_{i,j+1}^n - 2\theta_{i,j}^n + \theta_{i,j-1}^n}{(\Delta \eta)^2} \right) \dots \dots \dots (7)$$

Note the use of BD for the second term on the LHS and FD for the third term and CD for the RHS term.

The x-momentum equation can now be written in the finite difference form as

$$\frac{\bar{u}_{i,j}^{n+1} - \bar{u}_{i,j}^n}{\Delta \tau} + \bar{u}_{i,j}^n \frac{\bar{u}_{i,j}^n - \bar{u}_{i-1,j}^n}{\Delta \xi} + \bar{v}_{i,j}^n \frac{\bar{u}_{i,j+1}^n - \bar{u}_{i,j}^n}{\Delta \eta} = \theta_{i,j}^{n+1} + \frac{\bar{u}_{i,j+1}^n - 2\bar{u}_{i,j}^n + \bar{u}_{i,j-1}^n}{(\Delta \eta)^2} \dots \dots \dots (8)$$

The continuity equation becomes

$$\frac{\bar{u}_{i,j}^{n+1} - \bar{u}_{i-1,j}^{n+1}}{\Delta \xi} + \frac{\bar{v}_{i,j}^{n+1} - \bar{v}_{i,j-1}^{n+1}}{\Delta \eta} = 0 \dots \dots \dots (9)$$

Equations 7, 8 and 9 are solved sequentially for each time step. Note that θ^{n+1} is known once the energy equation is solved.

Therefore, in the momentum equation, it can be treated as a known quantity.

Since u is known from solving the x-momentum equation, v^{n+1} are the only unknowns in Eq. (9).

Thus these set of equations can be solved explicitly.