

EQUATIONS AND CONSTANTS

CONSTANTS

$$g_c = 32.2 \text{ lbm-ft/lbf-sec}^2$$

$$K_{\text{air}} = 1.395$$

$$R = 53.34 \text{ lbf-ft/lbm- } ^\circ\text{R}$$

$$C_p \text{ air} = 0.240 \text{ BTU/lbm- } ^\circ\text{R}$$

$$C_p \text{ water at } 70\text{EF} = 0.998 \text{ BTU/lbm- } ^\circ\text{F}$$

$$C_p \text{ water at } 160\text{EF} = 1.000 \text{ BTU/lbm- } ^\circ\text{F}$$

$$1 \text{ HP} = 42.4 \text{ BTU/min} = 0.746 \text{ Kwatts} = 550 \text{ ft-lbf/sec}$$

$$1 \text{ Watt} = 1 \text{ J/sec} = 0.05688 \text{ BTU/min}$$

$$1 \text{ BTU} = 778 \text{ ft-lbf}$$

$$1 \text{ in. H}_2\text{O} = 0.03611 \text{ lbf/in}^2 = 5.199840 \text{ lbf/ft}^2$$

$$1 \text{ in.hg.} = 0.491 \text{ lbf/in}^2$$

$$1 \text{ gal} = 0.1337 \text{ ft}^3$$

$$1 \text{ ton ref.} = 200 \text{ BTU/min}$$

$$1 \text{ BTU/lbm-EF} = 1 \text{ cal/g- } ^\circ\text{C}$$

$$1 \text{ slug} = 32.2 \text{ lbm}$$

$$T(^{\circ}\text{C}) = 5/9(^{\circ}\text{F}-32)$$

$$\rho \text{ g H}_2\text{O} = 62.4 \text{ lbf/ft}^3$$

$$T(^{\circ}\text{R}) = T(^{\circ}\text{F}) + 460$$

$$\text{air at STP} = 0.07654 \text{ lbm/ft}^3$$

$$T(^{\circ}\text{K}) = T(^{\circ}\text{C}) + 273$$

${}^\circ\text{F}$ = degrees Fahrenheit

${}^\circ\text{C}$ = degrees Celsius

${}^\circ\text{K}$ = degrees Kelvin

${}^\circ\text{R}$ = degrees Rankine

DYNAMIC BALANCING EQUATIONS

$$\Delta A_1 = \sqrt{\frac{(A_1^0)^2 + (A_1^{180})^2 - 2A^2}{2}} = \sqrt{\frac{(A_1^{90})^2 + (A_1^{270})^2 - 2A^2}{2}}$$

$$\Delta A_2 = \sqrt{\frac{(A_2^0)^2 + (A_2^{180})^2 - 2A^2}{2}} = \sqrt{\frac{(A_2^{90})^2 + (A_2^{270})^2 - 2A^2}{2}}$$

$$\Delta B_1 = \sqrt{\frac{(B_1^0)^2 + (B_1^{180})^2 - 2B^2}{2}} = \sqrt{\frac{(B_1^{90})^2 + (B_1^{270})^2 - 2B^2}{2}}$$

$$\Delta B_2 = \sqrt{\frac{(B_2^0)^2 + (B_2^{180})^2 - 2B^2}{2}} = \sqrt{\frac{(B_2^{90})^2 + (B_2^{270})^2 - 2B^2}{2}}$$

ψ = The angle of original unbalance

$$\psi = \tan^{-1} \left[\frac{A_1^{90})^2 - (\Delta A_1)^2 - A^2}{(A_1^0)^2 - (\Delta A_1)^2 - A^2} \right]$$

$$\psi_B = \tan^{-1} \left[\frac{(B_1^{90})^2 - (\Delta B_1)^2 - B^2}{(B_1^0)^2 - (\Delta B_1)^2 - B^2} \right]$$

$$\overline{M}_A = \overline{R}_A M_{trial}$$

$$\overline{M}_B = \overline{R}_B M_{trial}$$

$$\overline{R}_A \Delta A_1 + \overline{R}_B \Delta A_2 = -\overline{A}$$

$$\overline{R}_A \Delta B_1 + \overline{R}_B \Delta B_2 = -\overline{B}$$

$$R_{Ax} = \frac{B(\Delta A_2) \cos \psi_B - A(\Delta B_2) \cos \psi_A}{(\Delta A_2)(\Delta B_2) - (\Delta A_2)(\Delta B_1)}$$

$$R_{Bx} = \frac{-A \cos \psi_A - R_{Ax} \Delta A_1}{\Delta A_2}$$

$$R_{Ay} = \frac{B(\Delta A_2) \sin \psi_B - A(\Delta B_2) \sin \psi_A}{(\Delta A_1)(\Delta B_2) - (\Delta A_2)(\Delta B_1)}$$

$$R_{By} = \frac{-A \sin \psi_A - R_{Ay} \Delta A_1}{\Delta A_2}$$

VIBRATIONS/MECHATRONICS EXPERIMENT

Beam Data:

$$E = 69 GPa$$

$$\rho_b = 2700 \frac{kg}{m^3}$$

$$L_b = 30 cm$$

$$b = 0.0127 m \text{ (Width of the beam)}$$

$$h_b = 0.00317 m \text{ (Thickness of the beam in the bending direction)}$$

$$\beta_n L_b = 1.875$$

Equations:

$$f_n = \frac{(\beta_n L_b)^2}{2\pi L_b^2} \sqrt{\frac{EI}{\rho_l}}, \quad I = \frac{bh_b^3}{12}, \quad \rho_l = \rho_b b h_b$$

$$U_n(x) = \cosh(\beta_n x) - \cos(\beta_n x) + \left(\frac{B_2}{B_1} \right)_n (\sinh(\beta_n x) - \sin(\beta_n x))$$

$$\left(\frac{B_2}{B_1} \right)_n = \frac{\sin(\beta_n L_b) - \sinh(\beta_n L_b)}{\cos(\beta_n L_b) + \cosh(\beta_n L_b)}$$

$$f_{mass-added} \approx \frac{f_n}{\sqrt{1 + \frac{M U_n(x^*)^2}{M_b}}}$$

$$\delta_k = \ln \left(\frac{x_1}{x_k} \right), \quad \zeta = \frac{\delta_k}{\sqrt{\delta_k^2 + 4(k-1)^2 \pi^2}}$$

$$f_d = f_n \sqrt{1 - \zeta^2}$$

1-D TRIANGULAR FIN

1-D Fin Equation

$$\frac{d^2\theta}{dx^2} + \frac{1}{x} \cdot \frac{d\theta}{dx} - \frac{\theta}{x} \cdot p^2 = 0$$

where $\theta = T - T_\infty$, $\frac{x}{L} = x$, $L = \frac{L}{l}$, $Bi = h \cdot \frac{l}{k}$ and $p = \sqrt{BiL^2 f}$

$$x = 0, \quad \theta = \text{finite}$$

$$x = 1, \quad \theta = \theta_0 = T_w - T_\infty$$

$$\frac{\theta}{\theta_0} = \frac{I_0(2\sqrt{Bx})}{I_0(2\sqrt{B})} \quad \text{where } B = \frac{hl}{k} L^2 \sqrt{\frac{1}{L^2} + 1}$$

Least Squares Methodology

General

$$F_i = A \cdot f_i$$

$$\rho_i = F_i - (A \cdot f_i)$$

$$S = \sum \rho^2$$

$$= \sum f_i^2 - 2A \sum F_i f_i + \sum A^2 f_i^2$$

$$\frac{\partial S}{\partial A} = 0 \text{ results in}$$

$$A = \frac{\sum F_i f_i}{\sum f_i^2}$$

$$\sigma^2 = \frac{\sum (\rho_i - \bar{\rho})^2}{n}$$

$$\bar{\rho} = \frac{\sum \rho_i}{n}$$

Specific

$$\theta_i = D \cdot I_0(2\sqrt{Bx_i}) = D \cdot I_i$$

$$\rho_i = \theta_i - D \cdot I_i$$

$$S = \sum \rho^2$$

$$= \sum \theta_i^2 - 2D \sum \theta_i I_i + D^2 \sum I_i^2$$

$$\frac{\partial S}{\partial D} = 0 \text{ results in}$$

$$D = \frac{\sum \theta_i I_i}{\sum I_i^2} = \frac{\sum \theta_i I_0(2\sqrt{Bx_i})}{\sum I_0^2(2\sqrt{Bx_i})}$$

$$\sigma^2 = \frac{\sum (\rho_i - \bar{\rho})^2}{n}$$

$$\bar{\rho} = \frac{\sum \rho_i}{n}$$

ACOUSTICS

$$\rho = 1 - \alpha; c = f \lambda; S W R = 10^{\frac{\Delta SPL}{20}}, c^2_{\text{THEORY}} = K T R g_c, U = \frac{p}{\rho c}$$

$$p_t = p_i + p_r = A \sin \omega t + B \sin(\omega t - 2 \frac{\omega}{c} x - \theta)$$

$$|p_t| = \sqrt{A^2 + B^2 + 2AB \cos(2 \frac{\omega}{c} x + \theta)}$$

$$\alpha_n = \frac{A^2 - B^2}{A^2} = 1 - \frac{B^2}{A^2}$$

$$|p_t|_{MAX} = \sqrt{A^2 + B^2 + 2AB} = A + B$$

$$|p_t|_{MIN} = \sqrt{A^2 + B^2 - 2AB} = A - B$$

$$\alpha_n = \frac{4 |p_t|_{MAX} |p_t|_{MIN}}{(|p_t|_{MAX} + |p_t|_{MIN})^2}$$

$$L_{p,max} - L_{p,min} = 20 \log_{10} \left[\frac{|P_t|_{max}}{|P_t|_{min}} \right] dB$$

$$SWR = \frac{p_{MAX}}{p_{MIN}} = \frac{A + B}{A - B} = \frac{1 + B/A}{1 - B/A} = 10^{\frac{\Delta SPL}{20}}$$

$$\alpha_n = \frac{4 SWR}{(1 + SWR)^2}$$

PUMP EXPERIMENT

$$\dot{m} = \rho V A$$

$$H = \frac{1}{\rho g} (P_{out} - P_{in})$$

$$P_w = \rho g Q H$$

$$P_s = 2\pi N T$$

$$\eta = \frac{P_w}{P_s}$$

$$\eta_v = \frac{Q}{Q + Q_L}$$

$$u_d = \sqrt{u_o^2 + u_c^2}$$

$$u_y = \sqrt{\left(\frac{\partial f}{\partial x_1} u_{x_1}\right)^2 + \left(\frac{\partial f}{\partial x_2} u_{x_2}\right)^2}$$