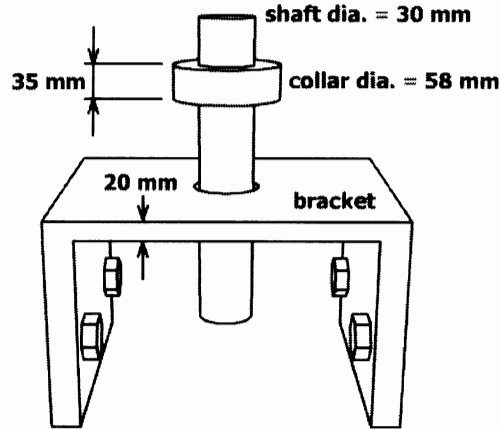
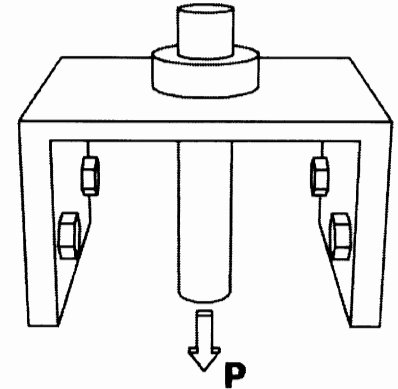


1. A shaft with a circular collar is inserted into the 30-mm-diameter hole in the bracket as shown. Determine the maximum axial force  $P$  that can be applied to the shaft if:

- (a) The allowable shear stress between the collar and shaft is 200 MPa, and  
 (b) The allowable bearing stress between the collar and the bracket is 450 MPa.



inserting shaft into bracket hole

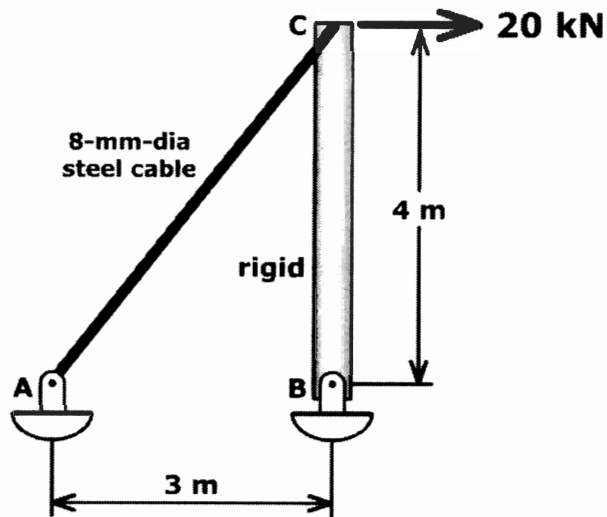


final position of shaft supported by bracket

$$200 \text{ MPa} = \frac{P}{\pi(0.03)(0.035)} \Rightarrow \boxed{P = 659.7 \text{ kN}}$$

$$450 \text{ MPa} = \frac{P}{\frac{\pi}{4}[(0.058)^2 - (0.03)^2]} \Rightarrow P = 870.8 \text{ kN}$$

2. The rigid vertical pipe BC is supported by a pin at B and the 8-mm-diameter steel cable AC. Assume  $E_s = 200 \text{ GPa}$  and  $\alpha_s = 12 \times 10^{-6} / ^\circ\text{C}$ .
- (a) Determine the change in length of member AC when the 20-kN load is applied and the temperature increases from  $+5^\circ\text{C}$  to  $+80^\circ\text{C}$ .
- (b) Determine the horizontal displacement of the connection at C.

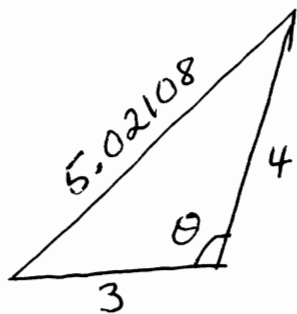


$$\sum M_B = 0 = AC \left( \frac{3}{5} \right) (4) - 20 \text{ kN} (4)$$

$$AC = 33.33 \text{ kN T}$$

$$a) \Delta_{AC} = \frac{(33,333)(5)}{\frac{\pi}{4}(1,008)^2(200 \times 10^9)} + (5)(12 \times 10^{-6})(75) = \boxed{0.02108 \text{ m}}$$

b)

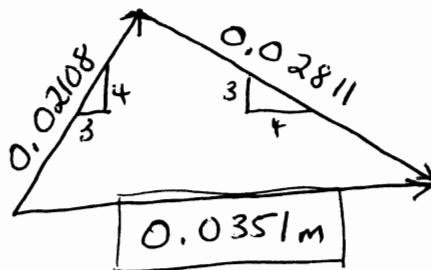


$$5.02108^2 = 3^2 + 4^2 - 2(3)(4) \cos \theta$$

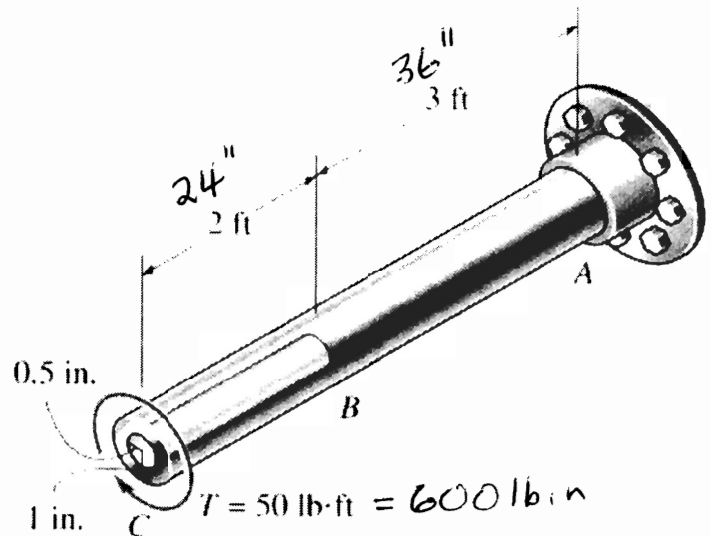
$$\theta = 90.5043^\circ$$

$$\Delta_c = 4 \sin(0.5043) = \boxed{0.0352 \text{ m} \rightarrow}$$

Small Displacement Approach



3. Section AB of the shaft is solid steel and section BC is a steel sleeve with a brass core. The entire shaft has a 2-inch outside diameter. The brass core in section BC has a diameter of 1 inch. The shaft is fixed at point A and a torque of 50 lb-ft is applied at end C. Determine the angle of twist at point C. Assume  $G_S = 11,000$  ksi and  $G_B = 6,000$  ksi.



$$\theta_{AB} = \frac{600(36)}{\frac{\pi}{32}(2)^4(11 \times 10^6)} = \underline{0.00125 \text{ rad}}$$

$$T_S + T_B = 600 \text{ lb-in}$$

$$\theta_S = \frac{T_S(24)}{\frac{\pi}{32}(2^4 - 1^4)(11 \times 10^6)} =$$

$$\theta_B = \frac{T_B(24)}{\frac{\pi}{32}(1)^4(6 \times 10^6)} =$$

$$\theta_S = \theta_B$$

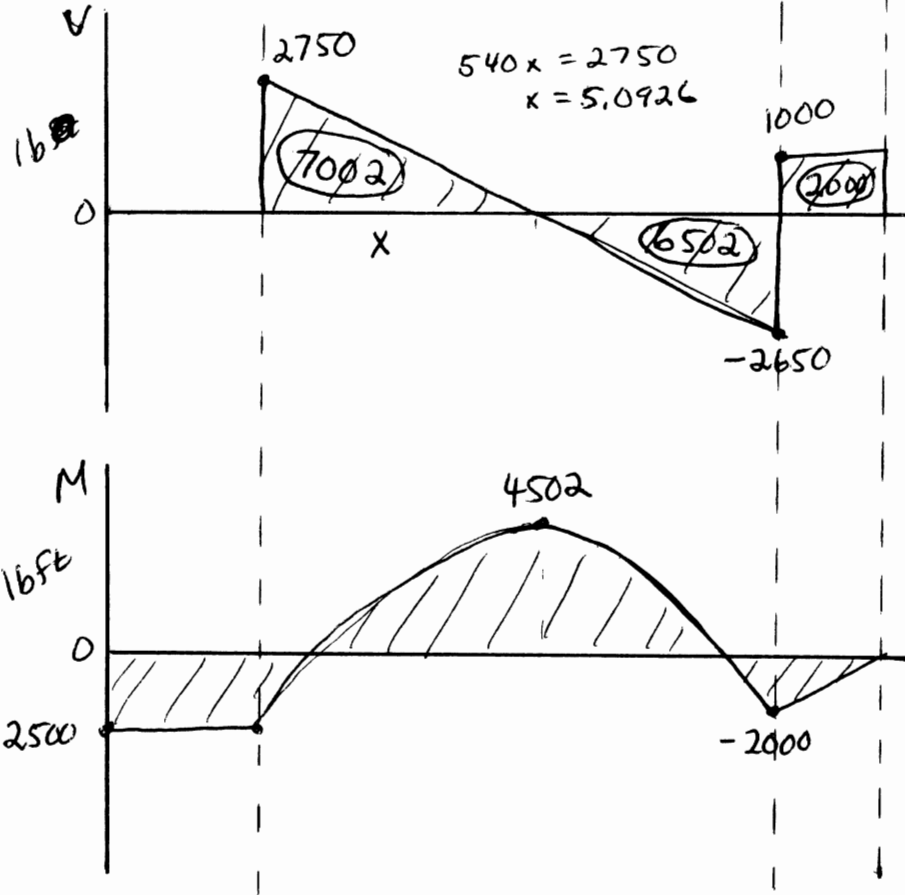
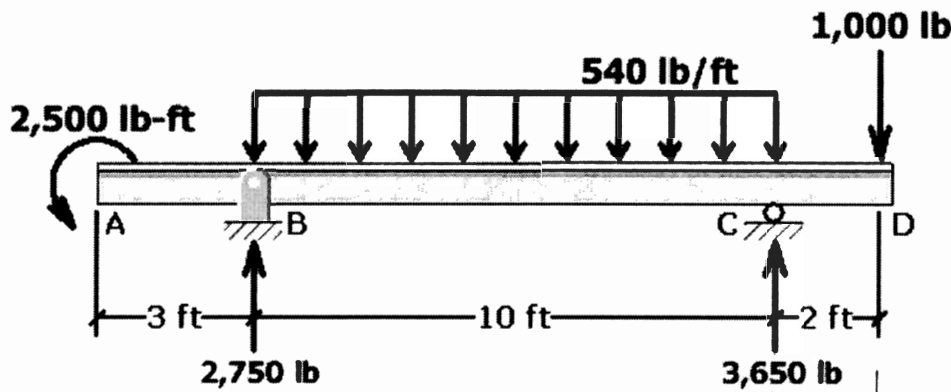
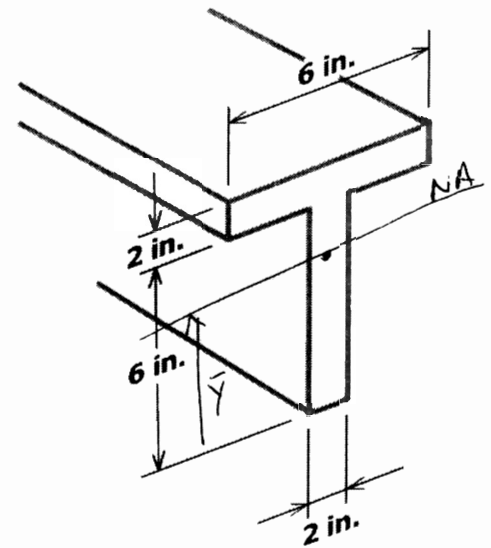
$$T_S = 578.9 \text{ lb-in}$$

$$T_B = 21.05 \text{ lb-in}$$

$$\theta_B = \theta_S = \underline{0.0008578 \text{ rad}}$$

$$\theta_C = \theta_{AB} + \theta_S = 0.00211 \text{ rad}$$

4. Determine the maximum tension and compression bending stresses at any location along the beam. The beam cross section is shown at the right. Support your answer.



$$\bar{y} = \frac{(2)(6)(3) + (2)(6)(7)}{2(2)(6)} = 5 \text{ in.}$$

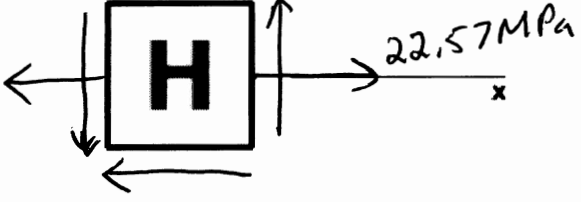
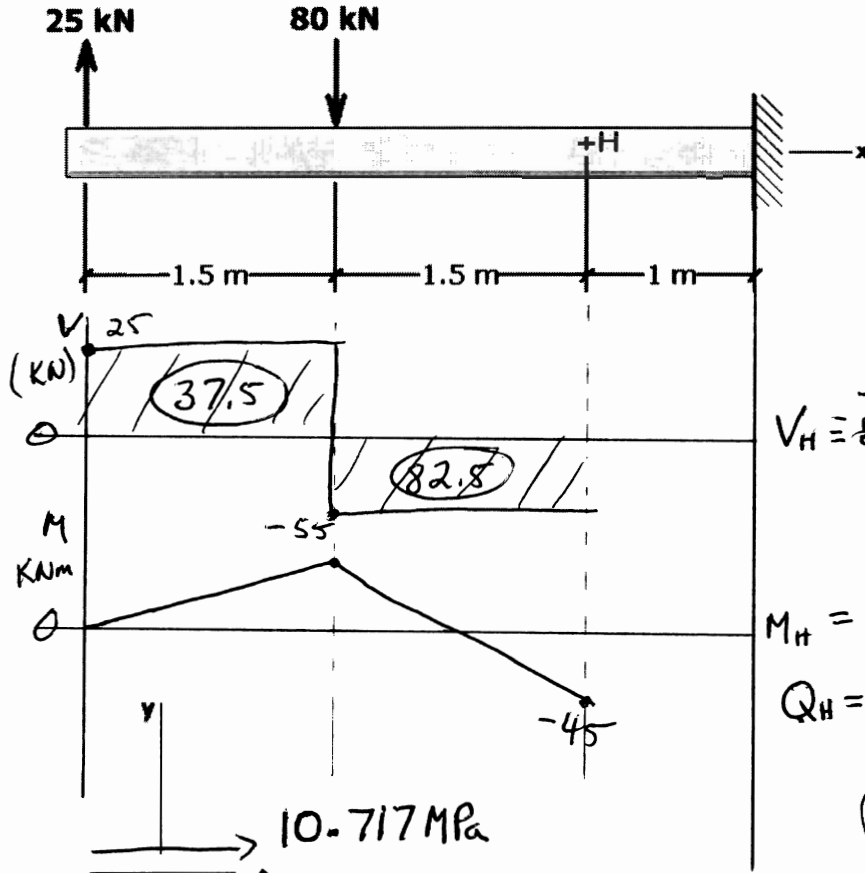
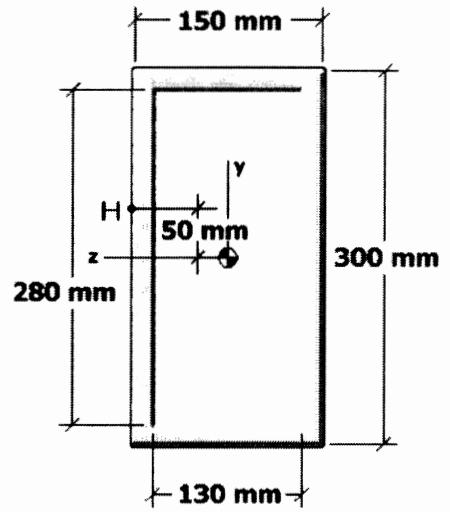
$$I = \frac{(2)(6)^3}{12} + (2)(6)(2)^2 + \frac{(6)(2)^3}{12} + (2)(6)(2)^2 = 136 \text{ in}^4$$

$4502 \cdot 3'' > 2500 \cdot 5''$   
 so:

$$\text{Max } T = \frac{4502 \cdot 12 \cdot 5}{136} = 1986 \text{ psi}$$

$$\text{Max } C = \frac{4502 \cdot 3 \cdot 12}{136} = 1192 \text{ psi}$$

5. Determine the bending stress and the transverse shear stress acting at point H in the beam shown. Point H is located 50 mm above the z centroidal axis of the box cross section. Show your results on the stress element below.



$$V_H = -55 \text{ kN}$$

$$I = \frac{150(300)^3}{12} - \frac{130(280)^3}{12}$$

$$= 99.69 \times 10^6 \text{ mm}^4$$

$$= 99.69 \times 10^{-6} \text{ m}^4$$

$$M_H = -45 \text{ kNm}$$

$$Q_H = (100)(150)(100) - (90)(130)(95) = 388,500 \text{ mm}^3$$

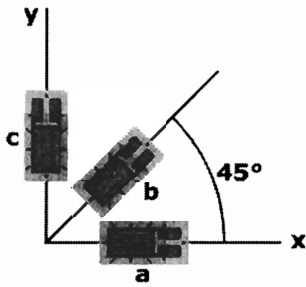
$$= 388.5 \times 10^{-6} \text{ m}^3$$

$$\tau_H = \frac{(45,000)(.05)}{99.69 \times 10^{-6}} = 22.57 \text{ MPa}$$

$$\sigma_H = \frac{(55,000)(388.5 \times 10^{-6})}{(99.69 \times 10^{-6})}$$

$$= 10.717 \text{ MPa}$$

6. The strain rosette shown was used to obtain the following normal strain data on the free surface of an aluminum plate ( $E = 70 \text{ GPa}$ ;  $\nu = 0.33$ ).  $\epsilon_a = +770 \mu\epsilon$ ;  $\epsilon_b = +1,180 \mu\epsilon$ ;  $\epsilon_c = -350 \mu\epsilon$ .



Determine:

- (a) The stress components  $\sigma_x$ ,  $\sigma_y$ , and  $\tau_{xy}$  at the point.
- (b) The principal stresses at the point.
- (c) Show the orientation of the principal stresses on an appropriate sketch.

$$\epsilon_x = 770 \mu \quad \epsilon_y = -350 \mu$$

$$1180 = 770 \cos^2 45 - 350 \sin^2 45 + \gamma_{xy} \sin 45 \cos 45$$

$$\gamma_{xy} = 1940 \mu$$

a)

$$\sigma_x = \frac{70 \times 10^9}{1 - 0.33^2} (770 \times 10^{-6} + (0.33)(-350 \times 10^{-6})) = 51.41 \text{ MPa}$$

$$\sigma_y = \frac{70 \times 10^9}{1 - 0.33^2} (-350 \times 10^{-6} + (0.33)(770 \times 10^{-6})) = -7.53 \text{ MPa}$$

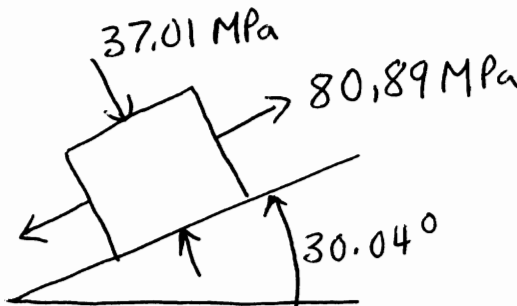
$$\tau_{xy} = \frac{70 \times 10^9}{2(1.33)} (1940 \times 10^{-6}) = 51.05 \text{ MPa}$$

b)

$$\sigma_{p1, p2} = \frac{51.41 - 7.53}{2} \pm \sqrt{\left(\frac{51.41 + 7.53}{2}\right)^2 + 51.05^2} = \{80.89, -37.01\} \text{ MPa}$$

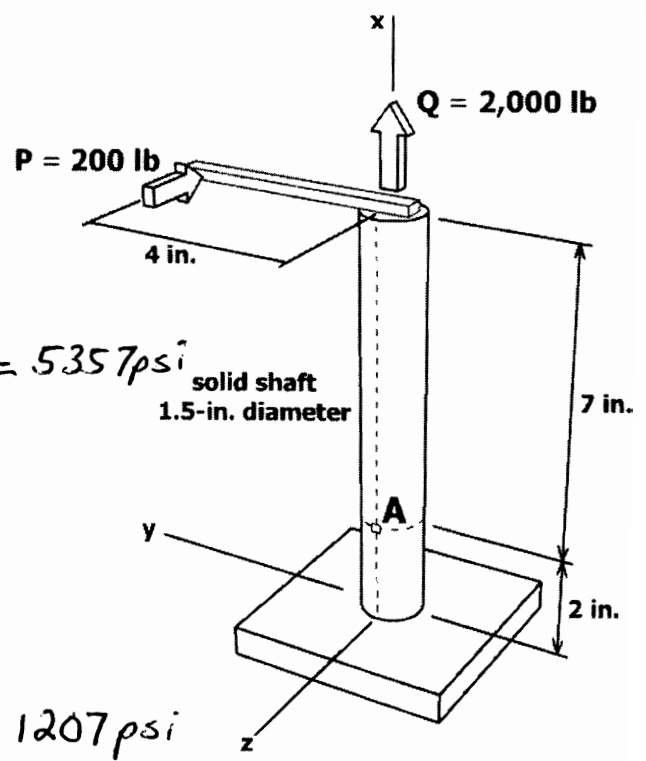
c)

$$\tan 2\theta_p = \frac{51.05}{\frac{51.41 + 7.53}{2}} \quad \theta_p = 30.04^\circ$$

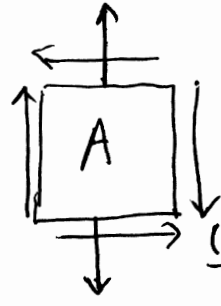


7. The 1.5-inch-diameter solid shaft is subjected to loads P and Q, as illustrated in the figure. Determine the maximum normal and shear stress at point A on the shaft. [Hint: Please consider the principal and in-plane maximum shearing stresses at A in finding your answer.]

B  
G  
E  
D  
H  
C  
A  
F



$$(5) \quad \frac{1132}{\frac{\pi}{4}(1.5)^2} + \frac{4225}{\frac{\pi}{64}(1.5)^4} = 5357 \text{ psi}$$

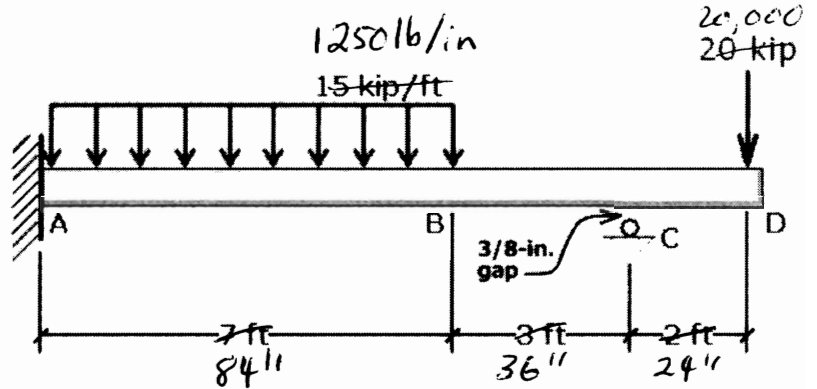


$$(6) \quad \frac{(200)(4)(.75)}{\frac{\pi}{32}(1.5)^4} = 1207 \text{ psi}$$

$$(4) \quad \sigma_{max} = \frac{5357}{2} + \sqrt{\left(\frac{5357}{2}\right)^2 + 1207^2} = \boxed{5616 \text{ psi}}$$

$$(4) \quad \tau_{max} = \sqrt{\left(\frac{5357}{2}\right)^2 + 1207^2} = \boxed{2938 \text{ psi}}$$

8. Determine the support reaction at C when there is a gap of 3/8-in. between the bottom of the beam and the top of the support before the beam is loaded. Assume  $EI = 5 \times 10^9 \text{ lb-in}^2$ .



$$\delta_c = -0.375 = \frac{R_c (120)^3}{3EI} - \frac{(20,000)(120)^2}{6EI} (3(144) - 120)$$

$$P = 20,000$$

$$x = 120$$

$$L = 144$$

$$- \frac{1250 (84)^4}{8EI} - \frac{1250 (84)^3}{6EI} (36)$$

$$R_c = 43,968 \text{ lb}$$