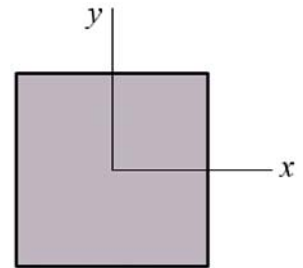
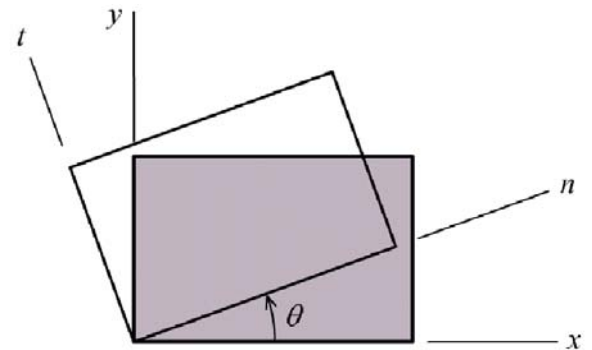


1. The strain components $\epsilon_x = 520 \mu\epsilon$, $\epsilon_y = -650 \mu\epsilon$, and $\gamma_{xy} = 750 \mu\text{rad}$ are given for a point in a body subjected to plane strain.

a. (4 points) Sketch the deformed shape on the element to the right.



b. (9 points) Determine the strain components ϵ_n , ϵ_t , and γ_{nt} at the point if the nt -axes are rotated with respect to the xy -axes by $\theta = 35^\circ$, as illustrated in the figure. Do not sketch the deformed shape of the element.



$\epsilon_n = \underline{\hspace{2cm}} \mu\epsilon$

$\epsilon_t = \underline{\hspace{2cm}} \mu\epsilon$

$\gamma_{nt} = \underline{\hspace{2cm}} \mu\text{rad}$

c. (14 points) Determine the angles θ_p and θ_s , principal strains, and maximum in-plane shear strain at the point. Do not sketch the deformed shapes of the element.

$\theta_p = \underline{\hspace{2cm}} \text{deg}$

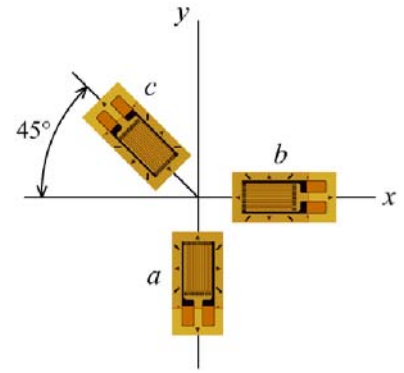
$\theta_s = \underline{\hspace{2cm}} \text{deg}$

$\epsilon_1 = \underline{\hspace{2cm}} \mu\epsilon$

$\epsilon_2 = \underline{\hspace{2cm}} \mu\epsilon$

$\gamma_{max} = \underline{\hspace{2cm}} \mu\text{rad}$

2. (10 points) The strain rosette shown in the figure was used to obtain normal strain data at a point on the free surface of a machine part. $\varepsilon_a = 550\mu$, $\varepsilon_b = -730\mu$, and $\varepsilon_c = -375\mu$. Determine the strain components ε_x , ε_y , and γ_{xy} at the point.



$$\varepsilon_x = \underline{\hspace{2cm}} \mu\varepsilon$$

$$\varepsilon_y = \underline{\hspace{2cm}} \mu\varepsilon$$

$$\gamma_{xy} = \underline{\hspace{2cm}} \mu\text{rad}$$

3. (8 points) The strain components $\varepsilon_x = 390\mu$, $\varepsilon_y = 820\mu$, and $\gamma_{xy} = -560\mu$ are given for a point on the free surface of a machine component. The modulus of elasticity for the material is $E = 73 \text{ GPa}$ and the Poisson's ratio is $\nu = 0.30$. Determine the stresses σ_x and τ_{xy} at the point.

$$\sigma_x = \underline{\hspace{2cm}} \text{ MPa}$$

$$\tau_{xy} = \underline{\hspace{2cm}} \text{ MPa}$$

4. (5 points) A spherical gas-storage tank with an inside diameter of 12 m is being constructed to store gas under an internal pressure of 1.75 MPa. The tank will be constructed from structural steel that has a yield strength of 250 MPa. If a factor of safety of 3.0 with respect to the yield strength is required, determine the minimum wall thickness required for the spherical tank.

$$t_{\min} = \underline{\hspace{2cm}} \text{ mm}$$

5. (5 points) A cylindrical boiler with an outside diameter of 3.60 m and a wall thickness of 40 mm is made of a steel alloy that has a yield stress of 415 MPa. Determine the maximum normal stress produced by an internal pressure of 2 MPa.

$$\sigma_{\max} = \underline{\hspace{2cm}} \text{ MPa}$$

6. A closed cylindrical vessel contains a fluid at a pressure of 720 psi. Assume $\sigma_{\text{hoop}} = 22.32$ ksi and $\sigma_{\text{axial}} = 11.16$ ksi. Determine:



- a. (4 points) the absolute maximum shear stress on the outer surface of the cylinder.

$$\tau_{\text{abs max}} = \underline{\hspace{2cm}} \text{ ksi}$$

- b. (4 points) the absolute maximum shear stress on the inner surface of the cylinder.

$$\tau_{\text{abs max}} = \underline{\hspace{2cm}} \text{ ksi}$$

7. (9 points) Three loads are applied to the short rectangular post. The cross-sectional dimensions of the post are shown.

a. Using the positive sign convention shown, determine the following internal forces and moments acting on a x - z plane through points H and K .

$$F_x = \underline{\hspace{2cm}} \text{ kN}$$

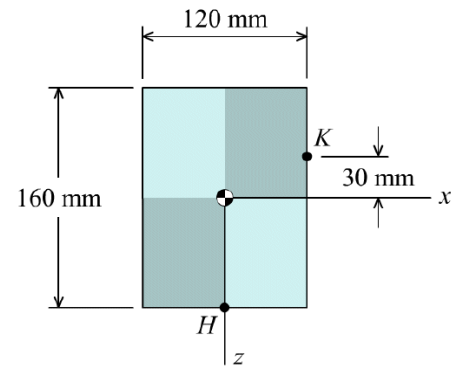
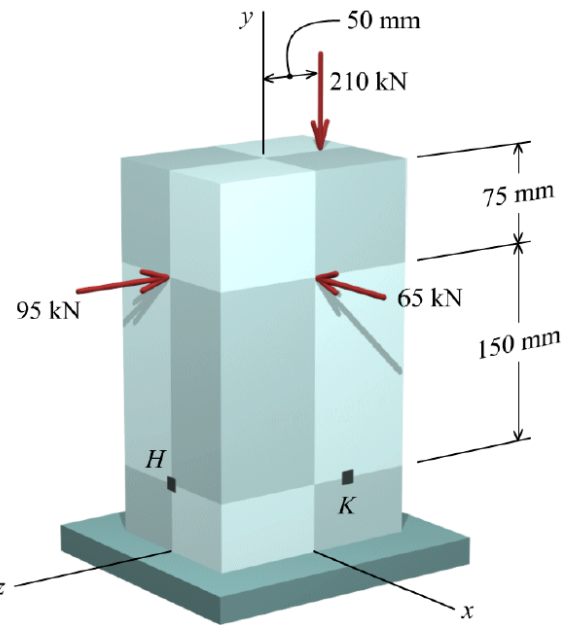
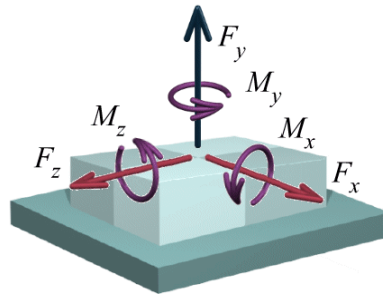
$$F_y = \underline{\hspace{2cm}} \text{ kN}$$

$$F_z = \underline{\hspace{2cm}} \text{ kN}$$

$$M_x = \underline{\hspace{2cm}} \text{ kN-m}$$

$$M_y = \underline{\hspace{2cm}} \text{ kN-m}$$

$$M_z = \underline{\hspace{2cm}} \text{ kN-m}$$



b. (14 points) For the following loadings, determine the normal and shear stresses at point H .

$$F_x = 48 \text{ kN}$$

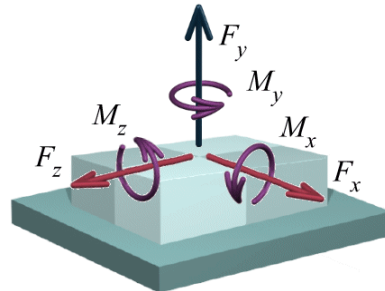
$$F_y = 0 \text{ kN}$$

$$F_z = 73 \text{ kN}$$

$$M_x = 3 \text{ kN-m}$$

$$M_y = 0 \text{ kN-m}$$

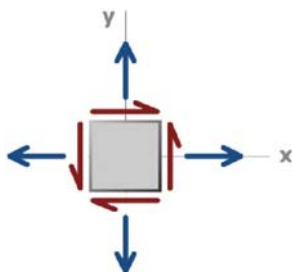
$$M_z = -2.5 \text{ kN-m}$$



$$\sigma_x = \underline{\hspace{2cm}} \text{ MPa}$$

$$\sigma_y = \underline{\hspace{2cm}} \text{ MPa}$$

$$\tau_{xy} = \underline{\hspace{2cm}} \text{ MPa}$$



8. (14 points) A solid steel crank has an outside diameter of 30 mm. For the following loadings, determine the normal and shear stresses on the top surface of the crank at point H .

$$F_x = 2350 \text{ N}$$

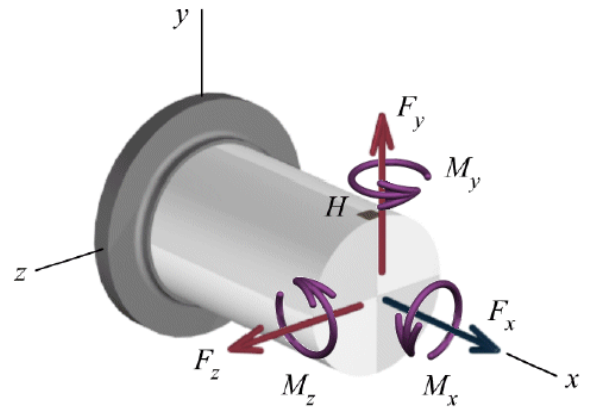
$$F_y = -1275 \text{ N}$$

$$F_z = 0 \text{ N}$$

$$M_x = 204 \text{ Nm}$$

$$M_y = 376 \text{ Nm}$$

$$M_z = 0 \text{ Nm}$$



$$\sigma_x = \text{_____ MPa}$$

$$\sigma_z = \text{_____ MPa}$$

$$\tau_{xz} = \text{_____ MPa}$$

