Name:
Section: 1A, 1B, 1C, 1D or Springfield

Select the best (closest) answer.

|  | a | b | c | d | e | value |
| ---: | :---: | :---: | :---: | :---: | :---: | :---: |
| 1. | O | O | O | O | O | 3 |
| 2. | O | O | O | O | O | 3 |
| 3. | O | O | O | O | O | 3 |
| 4. | O | O | O | O | O | 3 |
| 5. | O | O | O | O | O | 3 |
| 6. | O | O | O | O | O | 3 |
| 7. | O | O | O | O | O | 4 |
| 8. | O | O | O | O | O | 3 |
| 9. | O | O | O | O | O | 3 |
| 10. | O | O | O | O | O | 3 |
| 11. | O | O | O | O | O | 3 |
| 12. | O | O | O | O | O | 3 |
| 13. | O | O | O | O | O | 3 |
| 14. | O | O | O | O | O | 3 |
| 15. | O | O | O | O | O | 3 |
| 16. | O | O | O | O | O | 3 |


|  | a | b | c | d | e | value |
| :--- | :--- | :--- | :--- | :--- | :--- | :---: |
| 17. | O | O | O | O | O | 3 |
| 18. | O | O | O | O | O | 3 |
| 19. | O | O | O | O | O | 3 |
| 20. | O | O | O | O | O | 3 |
| 21. | O | O | O | O | O | 3 |
| 22. | O | O | O | O | O | 3 |
| 23. | O | O | O | O | O | 3 |
| 24. | O | O | O | O | O | 3 |
| 25. | O | O | O | O | O | 3 |
| 26. | O | O | O | O | O | 3 |
| 27. | O | O | O | O | O | 3 |
| 28. | O | O | O | O | O | 3 |
| 29. | O | O | O | O | O | 3 |
| 30. | O | O | O | O | O | 3 |
| 31. | O | O | O | O | O | 3 |
| 32. | O | O | O | O | O | 3 |
| 33. | O | O | O | O | O | 3 |

1. A stainless steel tube with an outside diameter of 70 mm and a wall thickness of 5 mm is used as a compression member. If the axial stress in the member must be limited to 340 MPa , determine the maximum load $P$ that the member can support.
a. 398 kN
b. 170 kN
c. 268 kN
d. 448 kN
e. 347 kN
2. Two 6-in.-wide wooden boards are to be joined by splice plates that will be fully glued on the contact surfaces. The glue to be used can safely provide a shear strength of 120 psi. Determine the smallest allowable length $L$ that can be used for the splice plates for an applied load of $P=15,000 \mathrm{lb}$. Note that a
 gap of 0.5 in. is required between boards (1) and (2).
a. 19.7 in .
b. 15.7 in.
c. 24.3 in.
d. 21.3 in.
e. 11.6 in.
3. A rigid bar $A B C D$ is supported by two bars as shown in the figure. There is no strain in the vertical bars before load $P$ is applied. After load $P$ is applied, the normal strain in rod (1) is $-1,400 \mu \mathrm{~m} / \mathrm{m}$. Determine the normal strain in rod (2).
a. $1650 \mu \varepsilon$
b. $2400 \mu \varepsilon$
c. $2850 \mu \varepsilon$
d. $2100 \mu \varepsilon$
e. $3150 \mu \varepsilon$

4. A large cement kiln has a length of 225 ft and a diameter of 8 ft . Determine the change in diameter of the structural steel $\left[\alpha=6.5 \times 10^{-6} /{ }^{\circ} \mathrm{F}\right]$ shell caused by an increase in temperature of $250^{\circ} \mathrm{F}$.
a. 0.195 in .
b. 0.156 in.
c. 0.328 in.
d. 0.273 in.
e. 0.410 in .
5. A steel plate is to be attached to a support with three bolts. The cross-sectional area of the plate is $800 \mathrm{~mm}^{2}$ and the yield strength of the steel is 250 MPa . The ultimate shear strength of the bolts is 475 MPa . A factor of safety of 1.67 with respect to yield is required for the plate. A factor of safety of 4.0 with respect to the ultimate shear strength is required for the bolts. Determine the minimum bolt diameter required to develop the full strength of the plate. Note: consider only the gross crosssectional area of the plate - not the net area.
a. 25.9 mm
b. 20.7 mm

6. A steel $[E=200 \mathrm{GPa}]$ rod with a circular cross section is $10-\mathrm{m}$ long. Determine the minimum diameter $D$ required if the rod must transmit a tensile force of 30 kN without stretching more than 5 mm .
a. 12.0 mm
b. 18.0 mm
c. 19.5 mm
d. 21.9 mm
e. 16.9 mm
7. The assembly shown consists of a brass shell (1) fully bonded to a ceramic core (2). The brass shell $[E=15,000$ ksi, $\left.\alpha=9.8 \times 10^{-6} /{ }^{\circ} \mathrm{F}\right]$ has an outside diameter of 2.00 in . and an inside diameter of 1.25 in . The ceramic core $[E=42,000$ ksi, $\alpha=1.7 \times 10^{-6} /{ }^{\circ} \mathrm{F}$ has a diameter of 1.25 in . At a temperature of $60^{\circ} \mathrm{F}$, the assembly is unstressed. Determine the largest temperature increase that is acceptable for the assembly if the normal stress in the longitudinal direction of
 the brass shell must not exceed 29 ksi.
a. $244{ }^{\circ} \mathrm{F}$
b. $320^{\circ} \mathrm{F}$
c. $410^{\circ} \mathrm{F}$
d. $372{ }^{\circ} \mathrm{F}$
e. $282{ }^{\circ} \mathrm{F}$
8. A compound shaft consists of brass segment (1) and aluminum segment (2). Segment (1) is a solid brass shaft with an outside diameter of 0.875 in . and an allowable shear stress of $6,000 \mathrm{psi}$. Segment (2) is a solid aluminum shaft with an outside diameter of

9. A motor supplies 14 kW at 600 rpm to gear A of the drive system shown. Shaft (1) is a solid $50-\mathrm{mm}$-diameter aluminum [ $\mathrm{G}=28 \mathrm{GPa}$ ] shaft with a length of $L_{1}=600 \mathrm{~mm}$. Shaft (2) is a solid 40-mm-diameter steel [G = 80 GPa ] shaft with a length of $L_{2}=400 \mathrm{~mm}$. Shafts (1) and (2) are connected at flange C , and the bearings shown permit free rotation of the shaft.
Determine the rotation angle of gear D with respect to gear B.

a. 0.0748 rad
b. 0.0305 rad
c. 0.0628 rad
d. 0.0284 rad
e. 0.0523 rad
10. The composite shaft shown consists of a stainless steel tube (1) and a brass tube (2) that are connected at flange $B$ and securely attached to rigid supports at $A$ and $C$. Stainless steel tube (1) has an outside diameter of 2.00 in ., a wall thickness of 0.250 in., a length of $L_{1}$ $=40$ in., and a shear modulus of 12,500 ksi. Brass tube (2) has an outside diameter of 3.500 in., a wall thickness of 0.219 in., a length of $L_{2}=20$ in., and a shear modulus of $5,600 \mathrm{ksi}$. If a concentrated torque of $T_{B}=42$ kip-in. is applied to flange $B$, determine the torque magnitude in tube (1).

a. 8.946 kip-in.
b. 6.894 kip-in.
c. 2.855 kip-in.
d. $4.652 \mathrm{kip}-\mathrm{in}$.
e. 1.544 kip-in.
11. Use the graphical method to construct the bending-moment diagram and identify the magnitude of the largest moment (consider both positive and negative). The ground reactions and shear-force diagram are shown.
a. $46.8 \mathrm{kN}-\mathrm{m}$
b. $40.5 \mathrm{kN}-\mathrm{m}$
c. $65.6 \mathrm{kN}-\mathrm{m}$
d. $58.7 \mathrm{kN}-\mathrm{m}$
e. $50.2 \mathrm{kN}-\mathrm{m}$

12. For a beam with the cross-section shown, find the magnitude of the bending stress at point $A$. The moment of inertia about the z axis is $862.7 \mathrm{in}^{4}$, and the centroid of the section is located 5.167 in from the bottom of the beam.
a. 998 psi
b. 706 psi
c. 463 psi
d. 658 psi
e. 755 psi

13. A composite beam is constructed of a Southern pine [ $E=10 \mathrm{GPa}$ ] timber, 200 mm wide by 360 mm deep, that is reinforced on its lower surface by a steel [ $E=200 \mathrm{GPa}$ ] plate that is 150 mm wide by 12 mm thick. Find the distance to the centroid of the transformed section from the bottom of the beam.
a. 130 mm
b. 134 mm
c. 140 mm
d. 145 mm
e. 149 mm

14. The internal shear force $V$ at a certain section of a steel beam is 80 kN , and the moment of inertia is $64,900,000$ $\mathrm{mm}^{4}$. Determine the horizontal shear stress at point $H$, which is located $L=20 \mathrm{~mm}$ below the centroid.
a. 40.6 MPa
b. 42.6 MPa
c. 38.9 MPa
d. 41.9 MPa
e. 43.7 MPa

15. A $\mathrm{W} 410 \times 60$ shape is strengthened by adding two $250-\mathrm{mm}$ wide by $17-\mathrm{mm}$ thick cover plates to its flanges, as shown. Each cover plate is attached to its

a. $735,000 \mathrm{~mm}^{3}$
b. $680,900 \mathrm{~mm}^{3}$
c. $954,000 \mathrm{~mm}^{3}$
d. $789,400 \mathrm{~mm}^{3}$
e. $898,900 \mathrm{~mm}^{3}$


| Designation | Area <br> A | Depth d | Web thickness $t_{w}$ | Flange width $b_{f}$ | Flange thickness $t_{f}$ | $I_{x}$ | $S_{x}$ | $r_{x}$ | $I_{y}$ | $S_{y}$ | $r_{y}$ |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
|  | $\mathrm{mm}^{2}$ | mm | mm | mm | mm | $\begin{gathered} 10^{6} \\ \mathrm{~mm}^{4} \end{gathered}$ | $\begin{gathered} 10^{3} \\ \mathrm{~mm}^{3} \end{gathered}$ | mm | $\begin{gathered} 10^{6} \\ \mathrm{~mm}^{4} \end{gathered}$ | $\begin{gathered} 10^{3} \\ \mathrm{~mm}^{3} \end{gathered}$ | mm |
| W410×85 | 10800 | 417 | 10.9 | 181 | 18.2 | 316 | 1510 | 171 | 17.9 | 198 | 40.6 |
| $410 \times 75$ | 9480 | 414 | 9.65 | 180 | 16.0 | 274 | 1330 | 170 | 15.5 | 172 | 40.4 |
| $410 \times 60$ | 7610 | 406 | 7.75 | 178 | 12.8 | 216 | 1060 | 168 | 12.0 | 135 | 39.9 |
| $410 \times 46.1$ | 5890 | 404 | 6.99 | 140 | 11.2 | 156 | 773 | 163 | 5.16 | 73.6 | 29.7 |

16. For the beam loaded as shown, use the method of superposition to determine the beam deflection at point $H$. Assume that $E I=1.0 \times 10^{7} \mathrm{kips}-\mathrm{in} .{ }^{2}$ is constant.
a. 0.7115 in.
b. 0.9305 in.
c. 0.4032 in.
d. 0.6048 in.
e. 0.3024 in .

17. For the beam loaded as shown, use the method of superposition to determine the beam deflection at point $C$. Assume that $E$ and $I$ are constant. $E=29,000$ $\mathrm{ksi} ; I=600 \mathrm{in} .{ }^{4}$
a. -0.7282 in.
b. -0.3734 in.
c. -0.6068 in .
d. -0.9103 in.
e. -0.4551 in .

18. The beam shown consists of a $W$ $610 \times 140$ structural steel wide flange shape $\left[E=200 \mathrm{GPa} ; I=1,120 \times 10^{6}\right.$ $\left.\mathrm{mm}^{4}\right]$. Determine the magnitude of the reaction at $B$ if $w=75 \mathrm{kN} / \mathrm{m}$ and $P=160$ kN .
a. 516.7 kN
b. 584.7 kN

c. 686.6 kN
d. 832.8 kN
e. 799.6 kN
19. Consider a point in a structural member that is subjected to plane stress. Normal and shear stresses acting on horizontal and vertical planes at the point are shown. If $\sigma$ (sigma $)=15 \mathrm{ksi}$, determine the angle $\theta_{\mathrm{s}}$ corresponding to the orientation of the maximum in-plane shear stress.
a. $66.7^{\circ}$ or $-23.3^{\circ}$
b. $60.7^{\circ}$ or $-29.3^{\circ}$
c. $77.2^{\circ}$ or $-12.8^{\circ}$
d. $56.7^{\circ}$ or $-33.3^{\circ}$
e. $54.0^{\circ}$ or $-36.0^{\circ}$

20. Determine the maximum in-plane shear stress and the average normal stress values acting at the point in the previous problem.
a. $\tau_{\text {in-plane } \max }=37.9 \mathrm{ksi} ; \sigma_{\text {average }}=17.5 \mathrm{ksi}$
b. $\tau_{\text {in-plane } \max }=28.9 \mathrm{ksi} ; \sigma_{\text {average }}=2.50 \mathrm{ksi}$
c. $\tau_{\text {in-plane } \max }=65.4 \mathrm{ksi} ; \sigma_{\text {average }}=50.0 \mathrm{ksi}$
d. $\tau_{\text {in-plane } \max }=49.8 \mathrm{ksi} ; \sigma_{\text {average }}=32.5 \mathrm{ksi}$
e. $\tau_{\text {in-plane } \max }=84.1 \mathrm{ksi} ; \sigma_{\text {average }}=70.0 \mathrm{ksi}$
21. Identify the stress-element that best depicts the maximum in-plane shear stress and the average normal stress values found in the previous problem.

c.

d.


A Mohr's circle is shown for a point in a physical object that is subjected to plane stress. If 1 grid square $=1 \mathrm{ksi}$,
22. Determine the stress $\tau_{\mathrm{xy}}$.
a. 12 ksi
b. 30 ksi
c. 48 ksi
d. 6 ksi
e. 18 ksi
23. Determine the angle $\theta_{p}$.
a. $76.7^{\circ}$
b. $-13.3^{\circ}$
c. $63.4^{\circ}$
d. $31.7^{\circ}$
e. $13.3^{\circ}$

24. Determine the stress $\tau_{\text {in-plane max }}$.
a. 6.71 ksi
b. 20.12 ksi
c. 33.54 ksi
d. 53.67 ksi
e. 13.42 ksi
25. Determine the stress $\tau_{\text {absolute max }}$.
a. 6.71 ksi
b. 20.12 ksi
c. 13.42 ksi
d. 33.54 ksi
e. 53.67 ksi
26. The thin rectangular plate shown is uniformly deformed such that $\varepsilon_{x}=-475 \mu \varepsilon, \varepsilon_{y}=+750 \mu \varepsilon$, and $\gamma_{x y}=-1,000 \mu \mathrm{rad}$. Determine the normal strain $\varepsilon_{A C}$ along diagonal $A C$ of the plate.
a. $906.1 \mu \varepsilon$
b. $-215.9 \mu \varepsilon$
c. $-459.8 \mu \varepsilon$
d. $759.8 \mu \varepsilon$
e. $-142.7 \mu \varepsilon$


The strain components $\varepsilon_{x}=-235 \mu, \varepsilon_{y}=-835 \mu$, and $\gamma_{x y}=200 \mu \mathrm{rad}$ are given for a point in a body subjected to plane strain.
27. Determine the center $\varepsilon_{\text {center }}$ of the corresponding in-plane Mohr's circle.
a. $-643 \mu$
b. $-583 \mu$
c. $-618 \mu$
d. $-535 \mu$
e. $-568 \mu$
28. Determine the radius $R$ of the in-plane Mohr's circle.
a. $286 \mu$

b. $316 \mu$
c. $244 \mu$
d. $251 \mu$
e. $282 \mu$
29. The normal strain measured on the outside surface of a spherical pressure vessel is $800 \mu \varepsilon$. The sphere has an outside diameter of 54 in . and a wall thickness of 0.50 in., and it will be fabricated from an aluminum alloy [ $E=10,000 \mathrm{ksi} ; v=0.33$ ]. Determine the internal pressure in the vessel.
a. 506.9 psi
b. 281.6 psi
c. 394.3 psi
d. 337.9 psi
e. 450.6 psi
30. The pressure tank shown is fabricated from spirally-wrapped metal plates that are welded at the seams in the orientation shown. The tank has an inside diameter of 500 mm and a wall thickness of 6 mm . For a gage pressure of 2.0 MPa, determine the normal stress perpendicular to the weld.

a. 29.44 MPa
b. 88.32 MPa
c. 95.68 MPa
d. 58.88 MPa
e. 73.60 MPa

A tee-shaped flexural member is subjected to an internal axial force of $P=4,000 \mathrm{~N}$, an internal shear force of $V=3,500 \mathrm{~N}$, and an internal bending moment of $M=2,000 \mathrm{~N}-\mathrm{m}$, as shown. If the centroid is 95 mm above the bottom edge of the tee-shape and the moment of inertia about the z -axis is $8,840,000 \mathrm{~mm}^{4}$, determine the following stresses at point $H$.
31. Normal stress $\sigma_{x}$.
a. 0.00 MPa
b. -1.99 MPa
c. -1.20 MPa
d. -1.66 MPa
e. -1.43 MPa
32. Normal stress $\sigma_{y}$.
a. -1.99 MPa
b. -1.43 MPa
c. -1.20 MPa
d. 0.00 MPa
e. -1.66 MPa
33. Shear stress $\tau_{\mathrm{xy}}$.

a. -1.46 MPa
b. -2.32 MPa
c. -1.77 MPa
d. -1.01 MPa
e. -2.02 MPa

## Key

1. e
2. d
3. d
4. b
5. b
6. c
7. d
8. c
9. b
10. b
11. e 12. a
12. a
13. b
14. e
15. d
16. c
17. c
18. c
19. b
20. e
21. d
22. d
23. a
24. a
25. b
26. d
27. b
28. e
29. d
30. e
31. d
32. с
